

N-S Eqs examples

<https://www.youtube.com/watch?v=xLQNqwPUuN4>

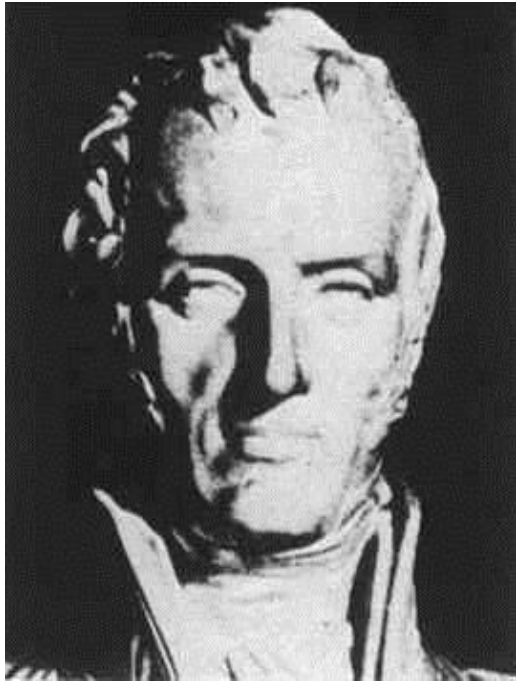
<https://www.youtube.com/watch?v=pVLCmT5lkw4&t=17s>

Development of Navier-Stokes equation

- Derived by Claude-Louis-Marie **Navier** in **1827**,
- and independently by Siméon-Denis **Poisson** in 1831.
 - Their motivations of the stress tensor were based on what amounts to a molecular view of how stresses are exerted by one fluid particle against another.
- Later, Barré **de Saint Venant** (in 1843)
- George Gabriel **Stokes** (in 1845) derived the equation starting with the **linear** stress rate-of-strain argument.

Development of Navier-Stokes equation

Louis Marie Henri Navier (1827)



George Gabriel Stokes (1845)



The Development of the Equation

- **Bernoulli** adapted methods of calculus to analyze fluid motion when subjected to various forces.
- **Euler** formulated a set of equations, which combined solutions describe precisely the motion of a viscosity-free fluid.
- **Navier** amended Euler's equations to account for viscosity.
- **Stokes rediscovered** Navier's equations, with proper mathematical reasoning.

N-S eqs

- **Newtonian, constant viscosity and constant density**

$$\rho \frac{D\vec{V}}{Dt} = \rho \vec{g} - \nabla \vec{p} + \mu \nabla^2 \vec{V}$$

x-direction:

$$\rho \left(\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right) =$$
$$\rho g_x - \frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$$

y-direction:

$$\rho \left(\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right) =$$
$$\rho g_y - \frac{\partial p}{\partial y} + \mu \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right)$$

z-direction:

$$\rho \left(\frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} \right) =$$
$$\rho g_z - \frac{\partial p}{\partial z} + \mu \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right)$$

NAVIER-STOKES EQUATIONS

(NEWTONIAN,
INCOMPRESSIBLE
FLUID)

$$\rho \frac{Du}{Dt} = \rho g_x - \frac{\partial P}{\partial x} + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$$

$$\rho \frac{Dv}{Dt} = \rho g_y - \frac{\partial P}{\partial y} + \mu \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right)$$

$$\rho \frac{Dw}{Dt} = \rho g_z - \frac{\partial P}{\partial z} + \mu \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right)$$

MOMENTUM CHANGE

GRAVITY FORCE

PRESSURE FORCE

VISCOUS FORCES

$$\frac{D(\cdot)}{Dt} = \frac{\partial(\cdot)}{\partial t} + u \frac{\partial(\cdot)}{\partial x} + v \frac{\partial(\cdot)}{\partial y} + w \frac{\partial(\cdot)}{\partial z}$$

"SUBSTANTIAL" DERIVATIVE
DERIVATIVE FOLLOWING MOTION OF A FLUID ELEMENT

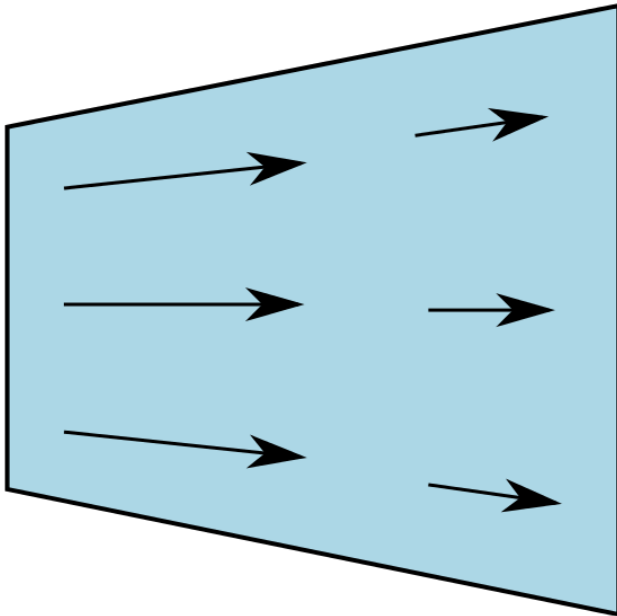
Local 'Temporal' Acceleration: velocity change with time

Convective Acceleration: velocity change with position – following the motion

Continuity equation

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) = 0$$

N-S Eqs – Convective Acceleration



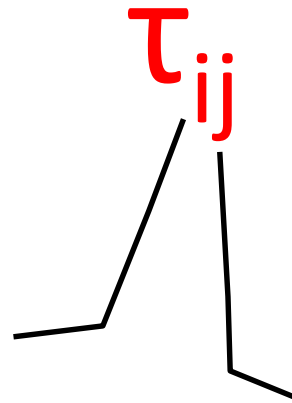
Though the flow may be steady (time-independent), the fluid decelerates as it moves down the diverging duct (assuming incompressible or subsonic compressible flow), hence there is an acceleration happening over position.

N-S eqs - **Nonlinearity**

- The nonlinearity is due to **convective acceleration**
- convective acceleration is associated with the change in velocity **over position**. Hence, any convective flow, whether turbulent or not, will involve nonlinearity.

Stress Tensor τ_{ij}

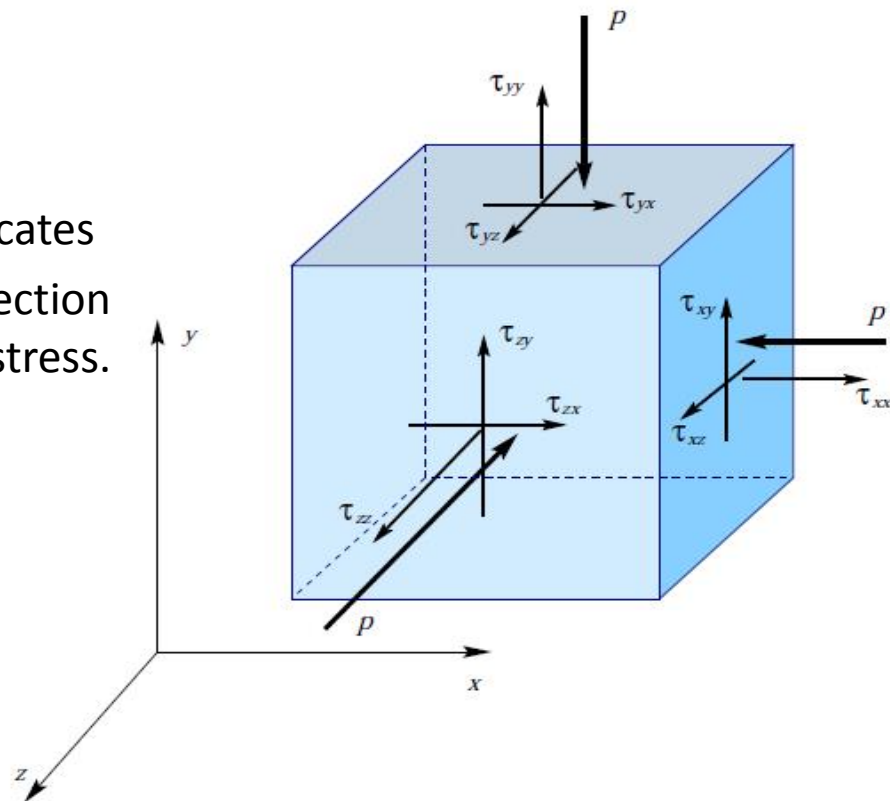
i indicates the face on which the stress is acting



j indicates the direction of the stress.

$$\sigma_{ii} = \tau_{ij}$$

normal stress on face i



N-S eqs: Alternative form

- Incompressible

$$u_t + uu_x + vv_y + ww_z = -\frac{1}{\rho}p_x + \nu\Delta u + \frac{1}{\rho}F_{B,x}, \quad (3.43a)$$

$$v_t + uv_x + vv_y + wv_z = -\frac{1}{\rho}p_y + \nu\Delta v + \frac{1}{\rho}F_{B,y}, \quad (3.43b)$$

$$w_t + uw_x + vw_y + ww_z = -\frac{1}{\rho}p_z + \nu\Delta w + \frac{1}{\rho}F_{B,z}. \quad (3.43c)$$

Here, ν is *kinematic viscosity*, the ratio of viscosity μ to density ρ , as given earlier in Chap. 2, and Δ is the second-order partial differential operator (given here in Cartesian coordinates)

$$\Delta = \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} + \frac{\partial^2}{\partial z^2}$$

known as the *Laplacian* or *Laplace operator* (which is usually denoted by ∇^2 in the engineering and

$$\underbrace{\underbrace{u_t}_{\text{local accel}} + \underbrace{uu_x + vu_y + wu_z}_{\text{convective accel}}}_{\text{total acceleration}} = \underbrace{-\frac{1}{\rho} p_x}_{\text{pressure forces}} + \underbrace{\nu \Delta u}_{\text{viscous forces}} + \underbrace{\frac{1}{\rho} F_{B,x}}_{\text{body forces}} .$$

N-S eqs - Viscous Forces

$$\nu \Delta u = \nu(u_{xx} + u_{yy} + u_{zz})$$

- Viscosity terms given above are associated with molecular transport (i.e., **diffusion**) of momentum.
- In general, second derivative terms in a differential equation are usually associated with **diffusion**, and in both physical and mathematical contexts this represents smoothing, or mixing process

High- and low-viscosity fluids

[lectures in elementary fluid dynamics - University of Kentucky](#)

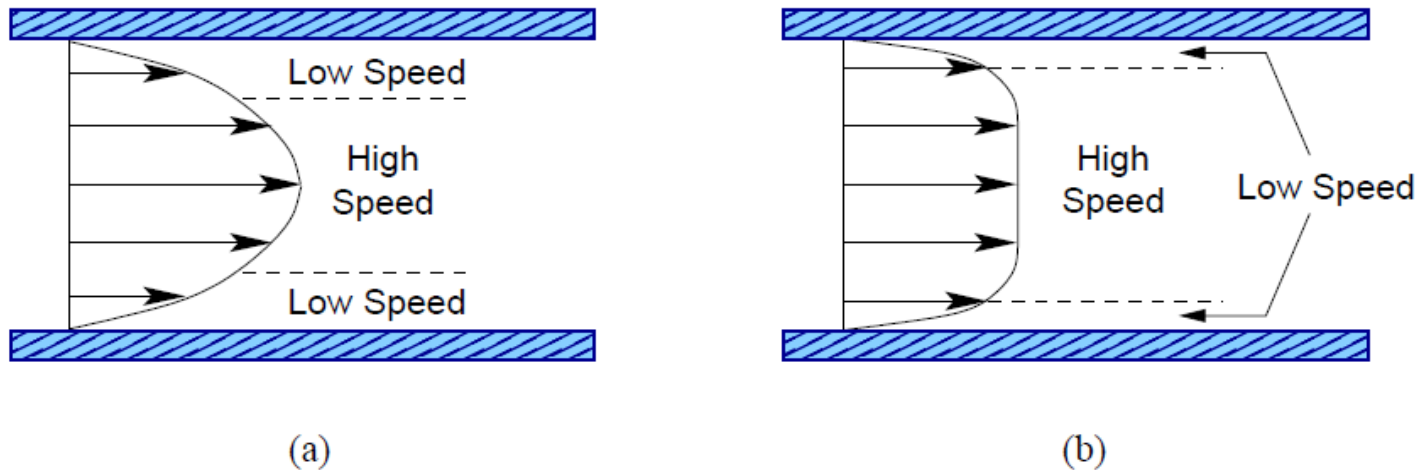


Figure 3.11: Comparison of velocity profiles in duct flow for cases of (a) high viscosity, and (b) low viscosity.

Viscous Forces – Boundary Layers

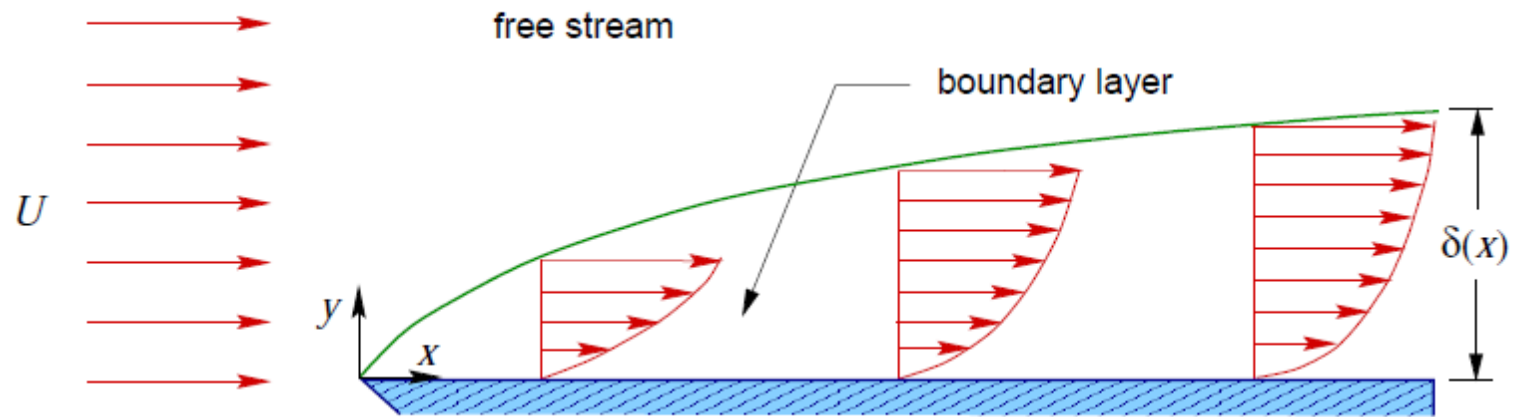
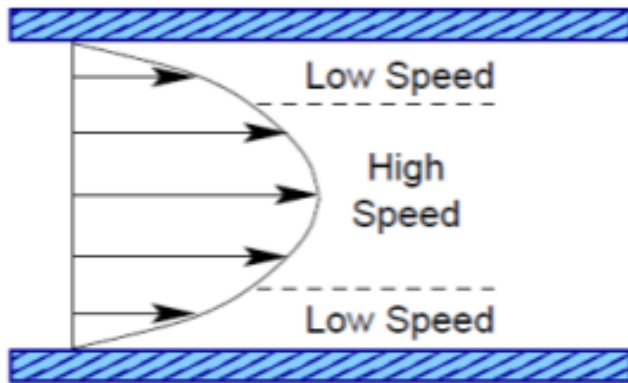


Figure 4.12: Steady, 2-D boundary-layer flow over a flat plate.

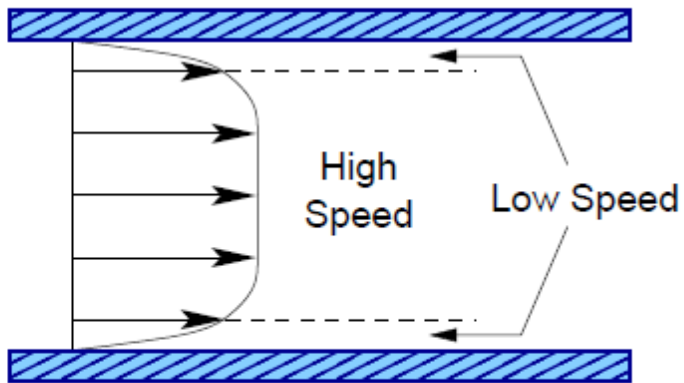
High viscosity fluids



(a)

- Velocity profile varies smoothly coming away from zero velocity at the wall, and reaching a maximum velocity in the center of the duct.
- Large viscosity: **diffusion** of viscous forces (time-rate of change of momentum) arising from high shear stress near the solid surfaces far into the flow field, thus smoothing the entire velocity profile

low-viscosity fluids



(b)

- narrow region of low-speed flow near the solid boundaries and a wider region of nearly constant-velocity flow in the central region of the duct
- speed on the centerline: is lower (for the same mass flow rate) than would be the high viscosity case.

Solution of N-S eqs

- **No** general analytical solution
 - **Millennium Prize Problems**
- Analytical solutions for few simple flow problems
- Complex flow problems:
 - Experimental investigation (cost, time) –
 - may be used to validate numerical solutions
 - Numerical solutions (magic alternative)

Millennium Prize Problems

- Seven problems in [mathematics](#) that were stated by the [Clay Mathematics Institute](#) in 2000.
- A correct solution to any of the problems results in a US \$1 million prize being awarded by the institute to the discoverer(s).
- The problems are [Birch and Swinnerton-Dyer conjecture](#), [Hodge conjecture](#), [Navier–Stokes existence and smoothness](#), [P versus NP problem](#), [Poincaré conjecture](#), [Riemann hypothesis](#), and [Yang–Mills existence and mass gap](#).
- At present, the only Millennium Prize problem to have been solved is the Poincaré conjecture, which was solved by the [Russian mathematician Grigori Perelman](#) in 2003.

Solution of N-S eqs

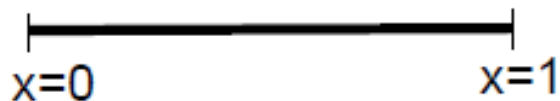
- Numerical solutions (magic alternative)
- <http://www.engr.uky.edu/~acfd/lecturenotes1.html>

The Strategy of CFD

Replace the continuous problem domain with a discrete domain using a grid.

Continuous Domain

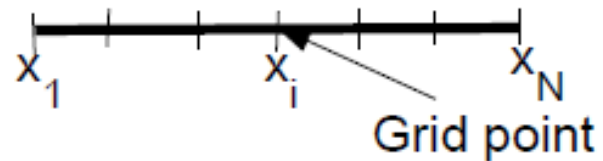
$$0 \leq x \leq 1$$



Coupled PDEs + boundary conditions in continuous variables

Discrete Domain

$$x = x_1, x_2, \dots, x_N$$

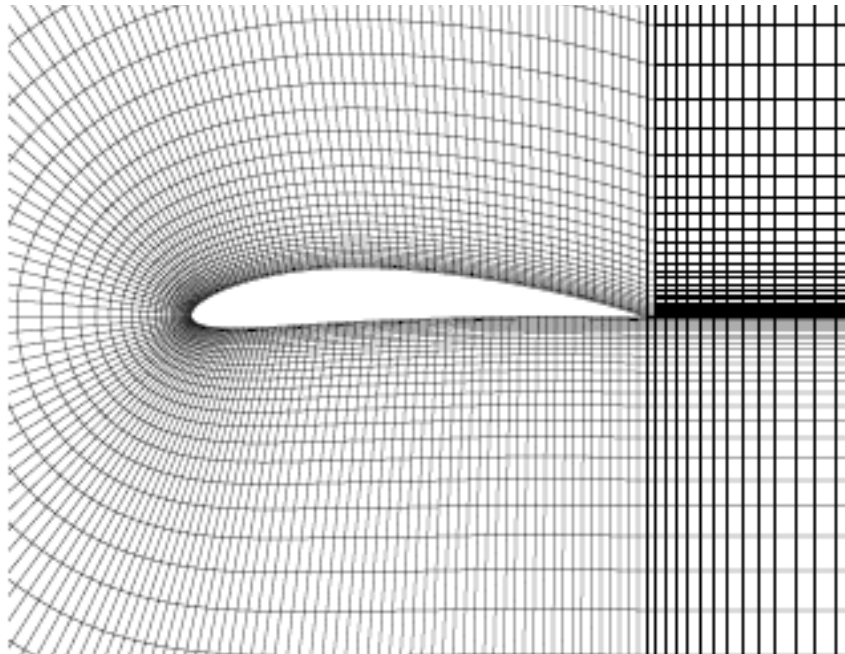


Coupled algebraic eqs. in discrete variables

CFD - discrete system

- The governing partial differential equations and boundary conditions are defined in terms of the continuous variables p , u , v , w etc
- The discrete system is a large set of **coupled algebraic equations** in the discrete variables.

CFD: Grid - Mesh



-
- **Parallel flow**[\[edit\]](#)
 - Assume steady, parallel, one dimensional, non-convective pressure-driven flow between parallel plates, the resulting scaled (dimensionless) [boundary value problem](#) is:
 - $\frac{d^2 u}{dy^2} = -1; u(0) = u(1) = 0.$ The boundary condition is the [no slip condition](#). This problem is easily solved for the flow field:
 - $u(y) = \frac{y - y^2}{2}.$ From this point onward more quantities of interest can be easily obtained, such as viscous drag force or net flow rate.


GENERAL SOLUTION PROCEDURE Navier-Stokes Eqs

① CHOOSE A COORDINATE SYSTEM

ASK YOURSELF: A) WHAT DIRECTION IS THE FLOW?

B) IN WHAT DIRECTION DOES THE VELOCITY CHANGE?

② DETERMINE THE DRIVING FORCE FOR FLOW (PRESSURE, SHEAR, GRAVITY)

③ WRITE THE BOUNDARY CONDITIONS (BCs) 

④ GUESS THE FORM OF THE SOLUTION → WHAT SHOULD IT LOOK LIKE?

⑤ SIMPLIFY THE CONSERVATION EQUATIONS

⑥ SOLVE THE RESULTING DIFFERENTIAL EQUATION

Couette flow

flow between two infinite parallel plates spaced a distance h apart in the y direction

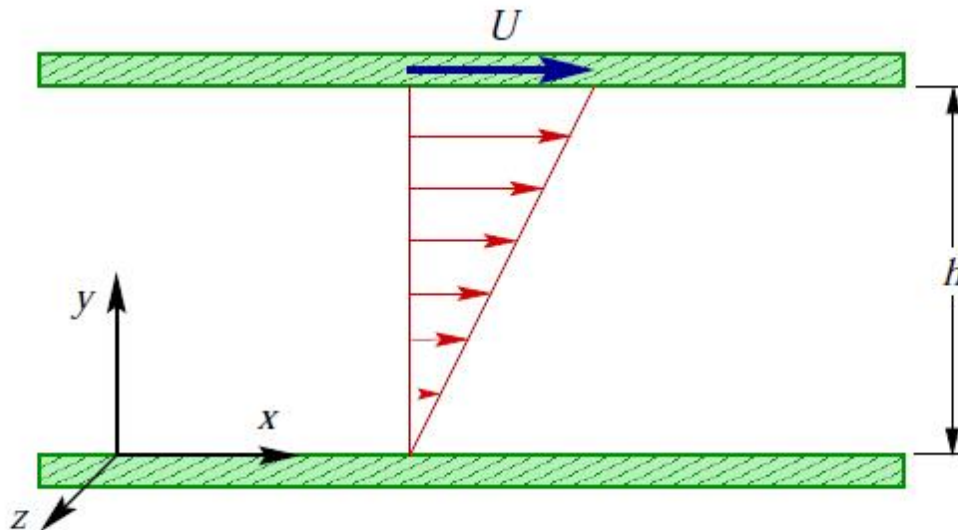


Figure 4.9: Couette flow velocity profile.

The Hagen–Poiseuille solution

- steady, incompressible, axisymmetric, fully-developed, laminar flow, $P_1 > P_2$

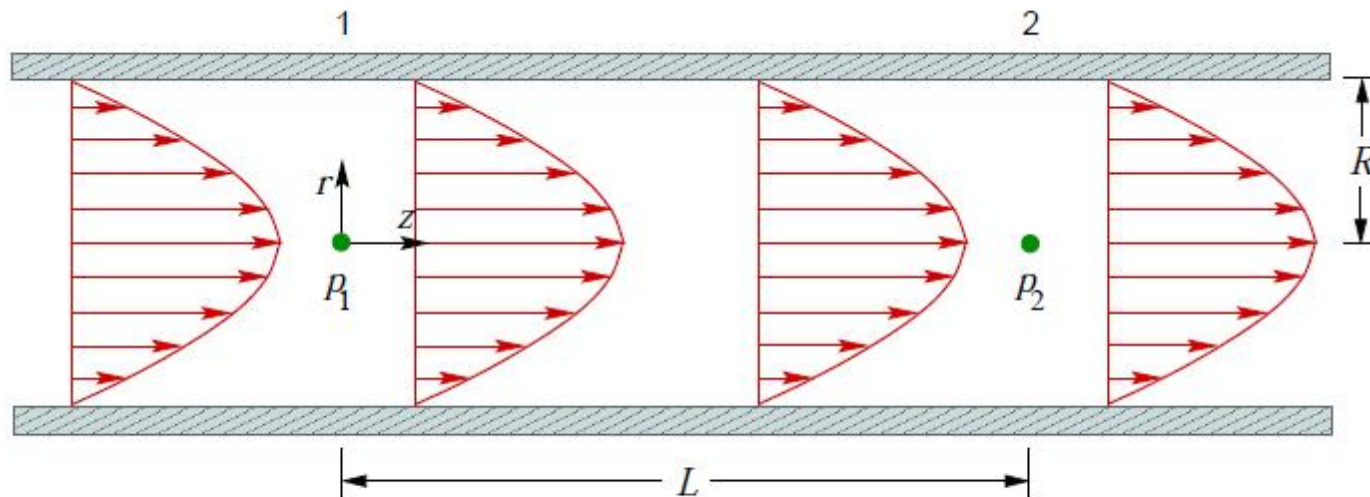


Figure 4.13: Steady, fully-developed pipe flow.

Ex: Possible flow field

- Use the continuity eq to check the possibility of the following steady incompressible flow with velocity field

$$u(x, y, z) = 2x + y + z, \quad v(x, y, z) = -y, \quad w(x, y, z) = -z.$$

Ex: Local & convective accelerations -1

Find the local and convective accelerations assuming a velocity field with the 3 components:

$$u = x + y + z + t, \quad v = x^2y^3zt, \quad w = \exp(xyzt)$$

Solution

$$\frac{DU}{Dt} = \frac{\partial U}{\partial t} + U \cdot \nabla U,$$

Ex: Local & convective accelerations -2

$$a_x \equiv \frac{Du}{Dt} = \frac{\partial u}{\partial t} + \mathbf{U} \cdot \nabla u,$$

$$a_x = u_t + uu_x + vu_y + wu_z,$$

$$a_y = v_t + uv_x + vv_y + wv_z,$$

$$a_z = w_t + uw_x + vw_y + ww_z.$$

Ex: Local & convective accelerations -3

$$u_t = 1, \quad u_x = 1, \quad u_y = 1, \quad u_z = 1,$$

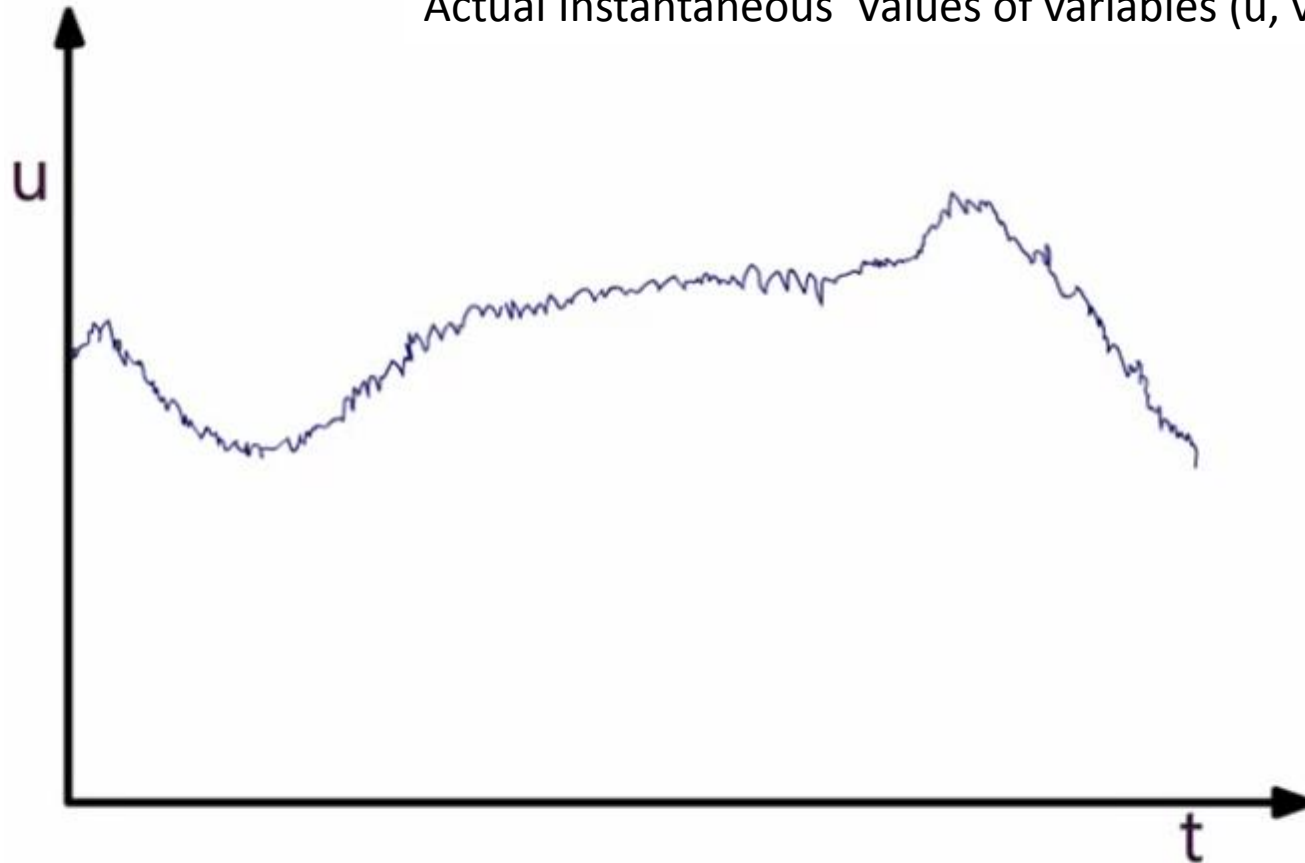
$$v_t = x^2 y^3 z, \quad v_x = 2xy^3 zt, \quad v_y = 3x^2 y^2 zt, \quad v_z = x^2 y^3 t,$$

$$w_t = xyz \exp(xyzt), \quad w_x = yzt \exp(xyzt),$$

$$w_y = xzt \exp(xyzt), \quad w_z = xyt \exp(xyzt).$$

Actual Instantaneous values

Actual Instantaneous Values of variables (u, v, w, p, ..)



Averaging process of N-S eqs

Actual velocity

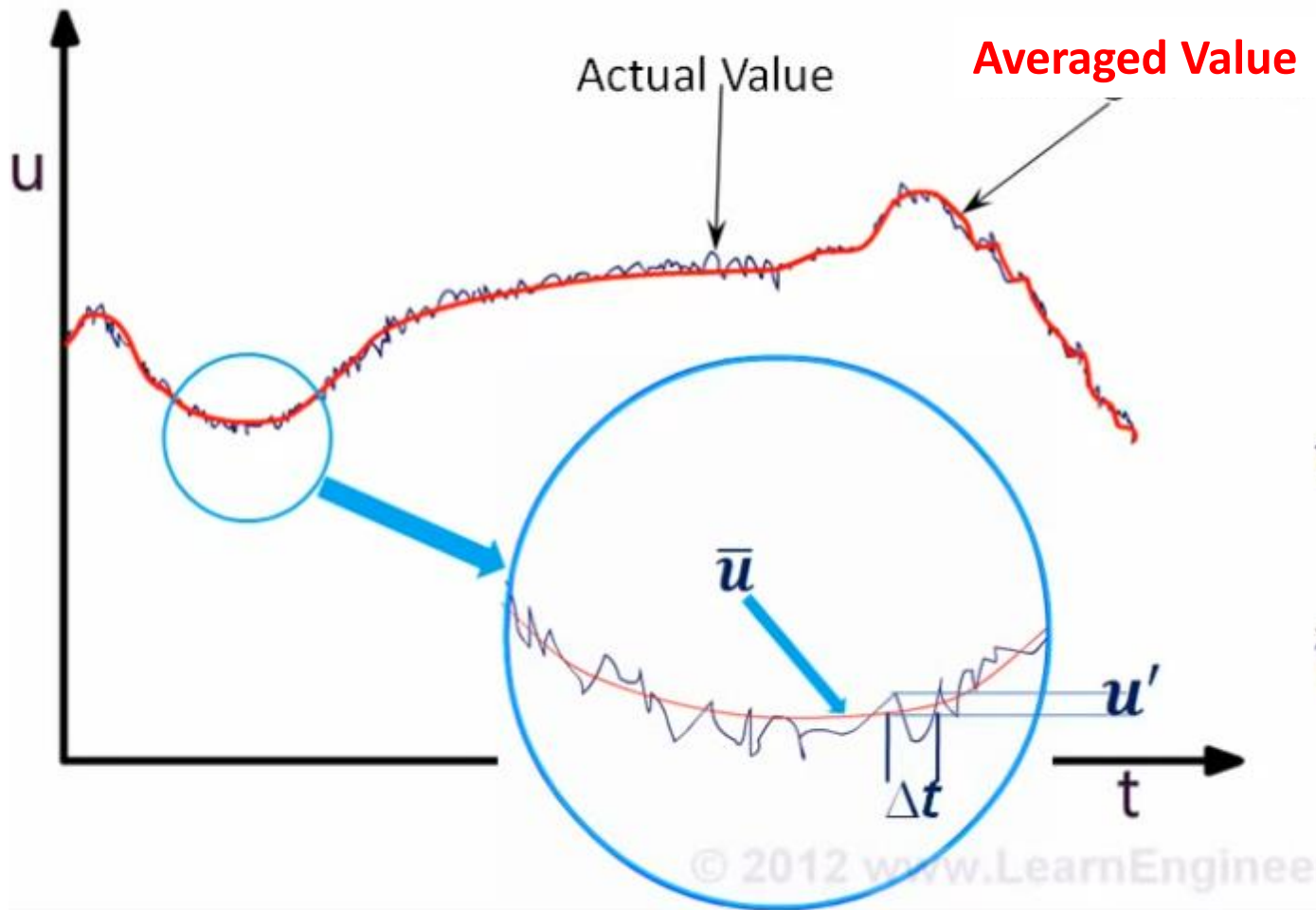
Average velocity

Fluctuating velocity

$$u = \bar{u} + u'$$

$$\bar{u} = \frac{1}{\Delta t} \int_t^{t+\Delta t} u dt$$

Averaging process of N-S eqs



RANS: Reynolds Average Navier Stokes



RANS

Reynolds Stress

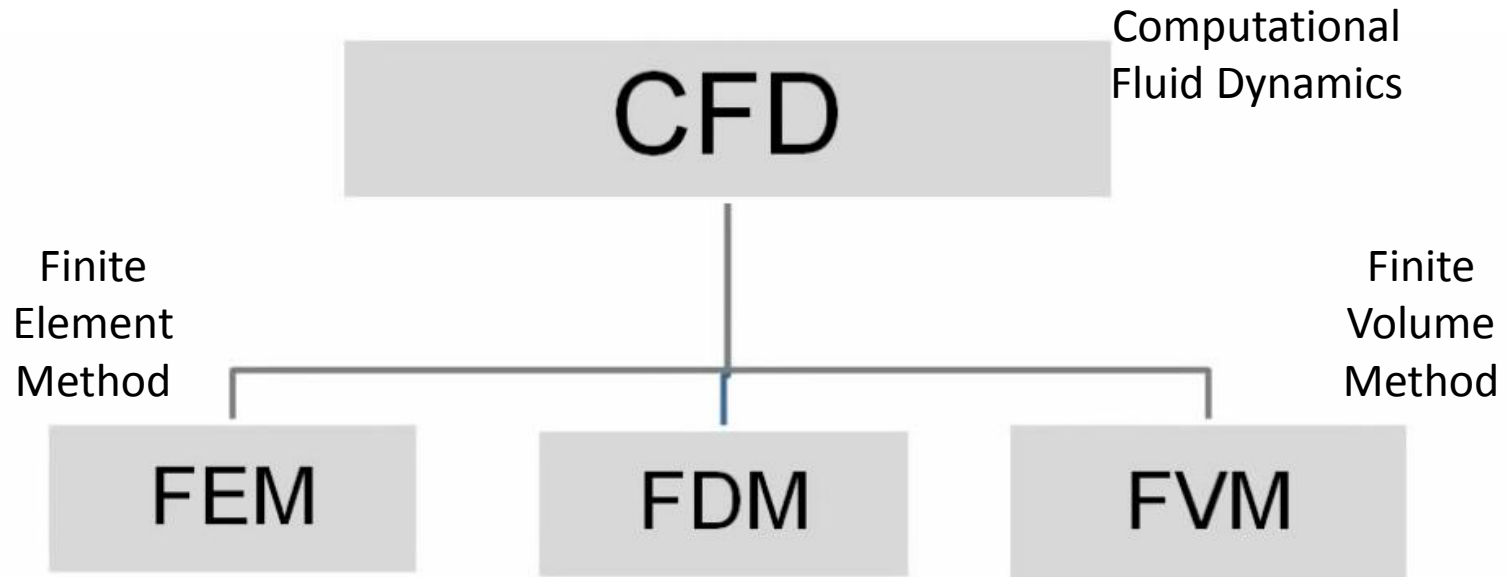
$$\rho \bar{u}_j \frac{\partial \bar{u}_i}{\partial x_j} = \rho \bar{f}_i + \frac{\partial}{\partial x_j} \left[-\bar{p} \delta_{ij} + \mu \left(\frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i} \right) - \overline{\rho u'_i u'_j} \right]$$

$$i, j = 1, 2, 3$$

$$\delta_{ij} = \begin{cases} 0 & i \neq j \\ 1 & i = j \end{cases}$$

Solving RANS

- RANS: Reynolds Average Navier Stokes



Finite
Difference
Method



AVL FIRE®



Numerical Solutions

N-S eqs: Nonlinear **PDEs** 'Partial Differential Equations' + BCs + ICs



Split Domain into grid / Mesh

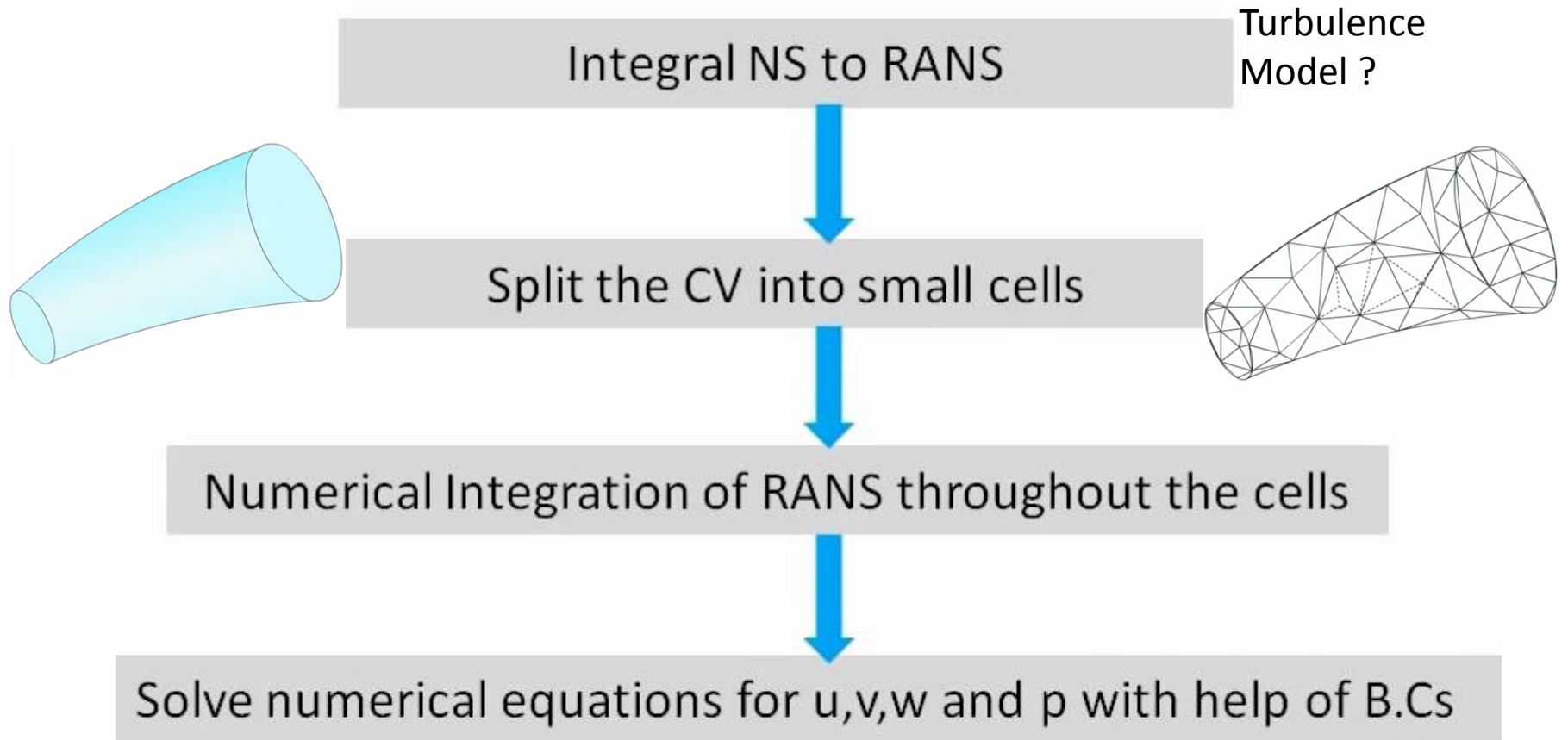


Discretization at mesh nodes:
Finite Difference – Finite Element – Finite Volume

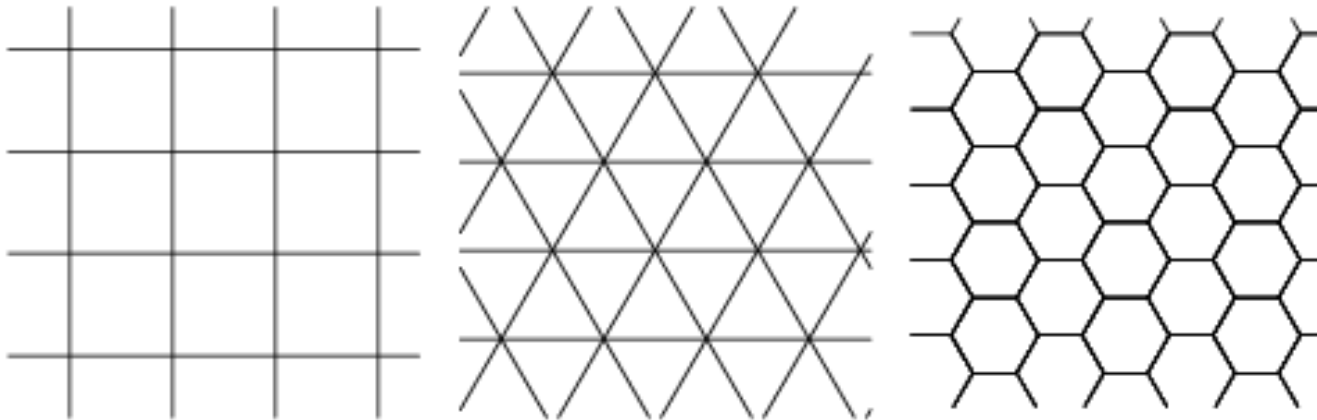


Set of simultaneous linear **algebraic** eqs

CFD FVM Steps



Grid / Mesh in 2-D

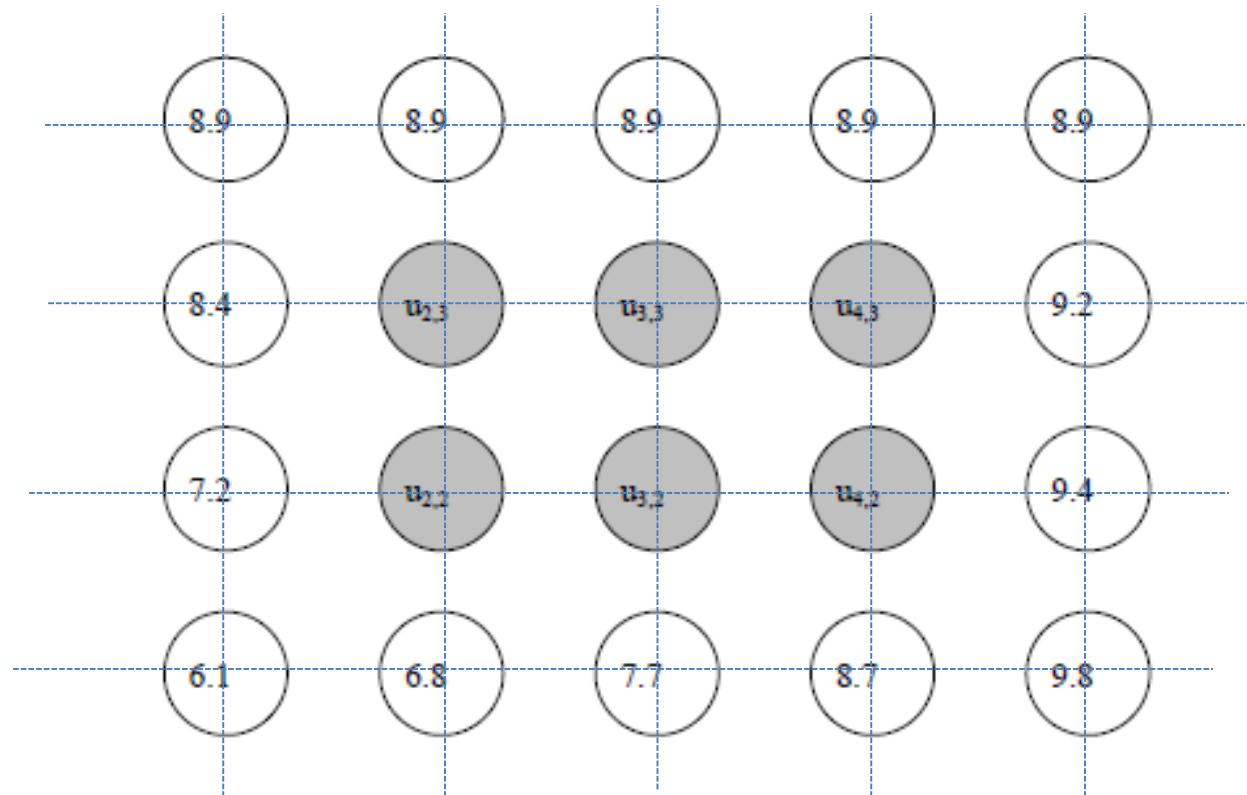


Discretization

partial derivative	finite difference approximation	type	order
$\frac{\partial U}{\partial x} = U_x$	$\frac{U_{i+1}^n - U_i^n}{\Delta x}$	forward	first in x
$\frac{\partial U}{\partial x} = U_x$	$\frac{U_i^n - U_{i-1}^n}{\Delta x}$	backward	first in x
$\frac{\partial U}{\partial x} = U_x$	$\frac{U_{i+1}^n - U_{i-1}^n}{2\Delta x}$	central	second in x
$\frac{\partial^2 U}{\partial x^2} = U_{xx}$	$\frac{U_{i+1}^n - 2U_i^n + U_{i-1}^n}{\Delta x^2}$	symmetric	second in x
$\frac{\partial U}{\partial t} = U_t$	$\frac{U_i^{n+1} - U_i^n}{\Delta t}$	forward	first in t
$\frac{\partial U}{\partial t} = U_t$	$\frac{U_i^n - U_i^{n-1}}{\Delta t}$	backward	first in t
$\frac{\partial U}{\partial t} = U_t$	$\frac{U_i^{n+1} - U_i^{n-1}}{2\Delta t}$	central	second in t
$\frac{\partial^2 U}{\partial t^2} = U_{tt}$	$\frac{U_i^{n+1} - 2U_i^n + U_i^{n-1}}{\Delta t^2}$	symmetric	second in t

Interior and Boundary Conditions

- Interior: Shaded
- Boundary: White



Finite Difference Method for Solving Elliptic PDE's

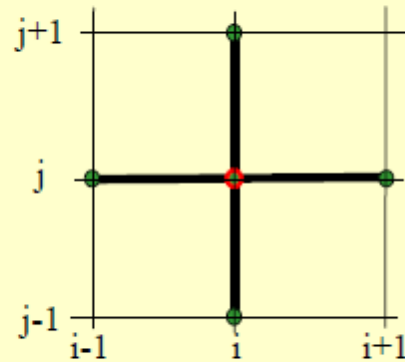
- Based on Boundary Conditions (BCs) and finite difference approximation to formulate system of equations
- Use Gauss-Seidel to solve the system

$$\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} = \begin{cases} 0 & \text{Laplace Eq.} \\ -D(x, y, u, \frac{\partial u}{\partial x}, \frac{\partial u}{\partial y}) & \text{Poisson Eq.} \end{cases}$$

-
1. Discretize domain into grid of evenly spaced points
 2. For nodes where u is unknown:
 $w / \Delta x = \Delta y = h$, substitute into main equation
 3. Using Boundary Conditions, write, $n * m$ equations for
 $u(x_{i=1:m}, y_{j=1:n})$ or $n * m$ unknowns.
 4. Solve this banded system

Laplace Eq

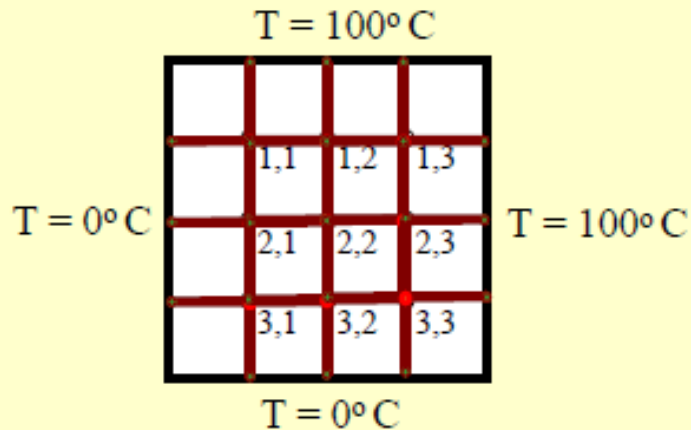
The Laplace molecule



If $\Delta x = \Delta y$ then

$$T_{i+1,j} + T_{i-1,j} + T_{i,j+1} + T_{i,j-1} - 4T_{i,j} = 0$$

Laplace Eq



The temperature distribution can be estimated by discretizing the Laplace equation at 9 points and solving the system of linear equations.

T_{11} T_{12} T_{13} T_{21} T_{22} T_{23} T_{31} T_{32} T_{33}

$$\begin{bmatrix}
 -4 & 1 & 0 & 1 & 0 & 0 & 0 & 0 & 0 \\
 1 & -4 & 1 & 0 & 1 & 0 & 0 & 0 & 0 \\
 0 & 1 & -4 & 0 & 0 & 1 & 0 & 0 & 0 \\
 1 & 0 & 0 & -4 & 1 & 0 & 1 & 0 & 0 \\
 0 & 1 & 0 & 1 & -4 & 1 & 0 & 1 & 0 \\
 0 & 0 & 1 & 0 & 1 & -4 & 0 & 0 & 1 \\
 0 & 0 & 0 & 1 & 0 & 0 & -4 & 1 & 0 \\
 0 & 0 & 0 & 0 & 1 & 0 & 1 & -4 & 1 \\
 0 & 0 & 0 & 0 & 0 & 1 & 0 & 1 & -4
 \end{bmatrix}
 \begin{Bmatrix}
 T_{11} \\
 T_{12} \\
 T_{13} \\
 T_{21} \\
 T_{22} \\
 T_{23} \\
 T_{31} \\
 T_{32} \\
 T_{33}
 \end{Bmatrix}
 =
 \begin{Bmatrix}
 -100 \\
 -100 \\
 -200 \\
 0 \\
 0 \\
 -100 \\
 0 \\
 0 \\
 -100
 \end{Bmatrix}$$

Excel
